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DLP® Series-450 DMD and System Mounting Concepts

1 Scope
As an aid to the successful first time utilization and implementation of the DLP5500, Series-450 DMD, this application report addresses the following topics:

- Terminology
- Specification and design details of a Series-450 DMD
- System mounting concepts for a Series-450 DMD, including key attributes and important application design considerations
- Sockets for use with a Series-450 DMD

2 Terminology

**Mechanical ICD** — The mechanical interface control drawing (ICD) describes the geometric characteristics of the DMD. This is also referred to as the Package Mechanical Characteristics.

**PCBoard** — Printed circuit board

**RSS** — Root sum square method of characterizing part tolerance stack-ups

**SUM** — Sum method of characterizing part tolerance stack-ups

**TP** — Thermal test point

**Optical Illumination Overfill** — The optical energy that falls outside the active area and does not contribute to the projected image.

**PGA** — Pin Grid Array (refers to a two-dimensional array of electrical contact pins)

**LGA** — Land Grid Array (refers to a two-dimensional array of electrical contact pads)

**Optical Assembly** — The sub-assembly of the end product which consists of the optical components and the mechanical parts that support those optical components

**Optical Chassis** — The main mechanical part used in the optical assembly to mount the DMD and optical components

**Optical Interface** — Refers to the features on the optical chassis used to align and mount the DMD

**Illumination Light Bundle** — Refers to the illumination cross-section area (size) at any location along the illumination light path but specifically at or near the DMD active array

**DMD Features** — The primary features of the Series-450 DMD are described below and illustrated in Figure 1 and Figure 2.

- DMD active array – the two-dimensional array of active DMD mirrors that reflect light
- WLP chip – Wafer Level Package (WLP) DMD chip that contains the DMD active array and window glass
- Window glass – the clear glass cover which protects the DMD active area (mirrors)
- Window aperture – the dark coating on the inside surface of the window around the active array
- Ceramic carrier – the structures which form the mechanical, optical, thermal, and electrical, interfaces between the WLP DMD chip and the end-application optical assembly
• Encapsulation – the material used to mechanically and environmentally protect the wire bond wires
• Bond wires – the wires which electrically connect the WLP DMD chip to the ceramic carrier
• Electrical pins – the electrical interface between the ceramic carrier and the end-application electronics
• Thermal interface area – the area on the ceramic carrier which allows direct contact of a heat sink or other thermal cooling device
• Corner chamfer – visual keying and orientation aid located on the ceramic carrier. Also identifies the incoming illumination direction
• Symbolization pad – area on the ceramic carrier where the human and bar code identification is located
• DMD chip (or just DMD) – The aggregate of the WLP chip, ceramic carrier, bond wires, encapsulation, and electrical pins
3 DMD Specifications

The key mechanical and thermal parameters of the DMD are described in this application note. The actual parameter values are specified in the DMD data sheet and Mechanical ICD. In case of any conflict between this document and either the data sheet or Mechanical ICD, the data sheet and Mechanical ICD should be used. (The Mechanical ICD is also referred to as the Package Mechanical Characteristics in the DMD data sheet.)
3.1 **DMD Documentation Structure**

The technical information for the DMD is contained in two sections of the data sheet, the basic part of the data sheet and the Mechanical ICD.

The overall size, datum locations, tolerances, and other geometric information are in the DMD Mechanical ICD. In some cases the Mechanical ICD may be a separate document from the basic part of the data sheet.

The functional characteristics and usage environment are in the DMD data sheet.

A 3D-CAD file of the DMD nominal geometry in STEP format is available for download, see Section 6.

Table 1 summarizes the content of each document.

<table>
<thead>
<tr>
<th>DMD Technical Information</th>
<th>Data Sheet</th>
<th>Mechanical ICD (Package Mechanical Characteristics)</th>
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<tbody>
<tr>
<td>Package geometry (dimensions, mounting datums, window thickness, window aperture size, active array size, etc.)</td>
<td></td>
<td>X</td>
</tr>
<tr>
<td>Thermal characteristics</td>
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</tr>
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<td>Mechanical mounting loads</td>
<td>X</td>
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</tr>
<tr>
<td>Optical properties (window material, mirror tilt angle, mirror size, etc…)</td>
<td>X</td>
<td></td>
</tr>
<tr>
<td>Electrical characteristics (signal names, voltage, wave form, etc…)</td>
<td>X</td>
<td></td>
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<tr>
<td>Operating environment</td>
<td>X</td>
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<tr>
<td>Storage environment</td>
<td>X</td>
<td></td>
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<tr>
<td>Part identification</td>
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</table>

3.2 **Optical Interface Features**

The Series-450 DMD incorporates three principle datum features (Datum ‘A’, Datum ‘B’, and Datum ‘C’). These datum features facilitate physical orientation of the DMD active array relative to the other optical components in the optical assembly. The dimensions and sizes of the datum features are defined in the Mechanical ICD drawing. The three datum features are shown in Figure 3 and described below.

Datum ‘A’ – Primary Datum

Datum ‘A’ is a plane specified by 3 areas on the surface of the ceramic carrier. The plane of the DMD active array is parallel to the plane formed by the three Datum ‘A’ areas. The DMD active array has a controlled distance and parallelism from Datum ‘A’, as defined in the Mechanical ICD. Datum ‘A’ allows the plane of the active array to be precisely (and repeatable) oriented along the system optical axis. The specific size and location of the areas defining Datum ‘A’ are specified in the Mechanical ICD.

Datum ‘B’ – Secondary Datum

Datum ‘B’ is a hole with a specified depth. While Datum ‘A’ defines the reference location of the active array plane axially along the system optical axis, Datum ‘B’ establishes the reference for the X and Y position of the active array within the Datum ‘A’ plane. Datum ‘B’ is not the entire depth of the hole in the ceramic but rather the top region closest to the Datum ‘A’ areas, see Figure 3. The location, size, and tolerance of the hole are defined in the Mechanical ICD.

Datum ‘C’ – Tertiary Datum

Datum ‘C’ is the center of a slot with a specified depth. Datum ‘C’ establishes the reference rotation of the active array within the Datum ‘A’ plane and about the Datum ‘B’ X-Y reference position. Datum ‘C’ is not the entire depth of the slot in the ceramic but rather the top region closest to the Datum ‘A’ areas, see Figure 3. The location, size, and tolerance of the slot are defined in the Mechanical ICD.
3.3 **DMD Cross Section Features**

Figure 4 illustrates the features of the DMD in cross-section. Shown are the window thickness, distance from active array to the window, window aperture location, ceramic carrier thickness, Datum 'A' plane location, active array plane, and encapsulation profile. The nominal distance and tolerance between these features are defined in the DMD Mechanical ICD.
3.4 System Dust Gasket and System Aperture

The flat area of the ceramic carrier which contains the Datum ‘A’ areas (but not including the Datum ‘A’ areas) can provide a resting (or mating) surface for a system optical aperture and/or dust gasket. The system aperture and gasket should be kept clear of the Datum ‘A’ areas, Datum ‘B’ hole, and Datum ‘C’ slot to ensure the proper optical alignment of the DMD is not interfered with. This surface is depicted in Figure 3.

As shown in Figure 4 the outside surface of the DMD window is relatively close to the plane of the DMD active array. Since the DMD active array will be an optical focus plane, there is a risk of particles on the outside window surface being reimaged to the projection plane. To prevent this from occurring it is best to prevent dust from getting onto the outside surface of the DMD window. This can be accomplished by placing a flexible dust gasket between the optical chassis and the gasket mounting area described previously. It is import that the gasket be flexible (compressive) enough that it does not interfere with the contact between the DMD Datum ‘A’ features and the associated features on the optical chassis.

The profile of the encapsulation around the window glass should be taken into consideration when sizing the openings and features of the optical engine, the dust gasket (if used), and the system aperture (if used) to ensure they do not interfere with the maximum allowed size of the DMD encapsulation.

3.5 Optical Illumination Overfill

Optical illumination overfill is defined as the optical energy that falls outside the active area, and which does not contribute to the projected image. The shape and distribution of the optical energy in the overfill region is determined by the system optical design. A possible illumination profile and associated overfill is illustrated in Figure 5.

Typical attributes that result in different overfill profiles include (but are not limited to) integrator size, illumination source, and optical aberrations (such as distortion and/or color separation).

Excess optical illumination overfill can result in high thermal loads on the DMD (which must be cooled by the system) and/or various types of image artifacts (e.g., stray light).

The magnitude of these effects depends upon several factors which include (but not limited to):

- The total amount of energy being reflected from the DMD active array
- The total amount of energy within the overfill area
- The spatial distribution of energy within the overfill area
- The specific DMD feature upon which overfill is incident (window aperture, dark area around the active array at the array plane), etc…
- The thermal management system used to cool the DMD
- The type of end-application (e.g., front projection display, rear projection display, etc…)
- The specific Series-450 DMD being used

To improve system optical efficiency, reduce the thermal cooling load, and reduce any possible optical artifacts the amount of energy outside the active array should be minimized.

Optical overfill energy on the window aperture (if present) should especially be avoided. The heat absorbed by the window aperture (due to overfill that is incident upon the window aperture) is more difficult to remove (more resistive thermal path) than heat absorbed in the dark area surrounding the active array.
3.6 **Active Array Size and Location**

The DMD Datum ‘A’, Datum ‘B’ hole, and Datum ‘C’ slot are the same for all DMDs in the Series-450 DMD family. While the active array size is different for each DMD resolution within the Series-450 family the center of the active array has the same location relative to the DMD Datum’s for all the Series-450 DMDs.

Note that the center of the active array is not at the center point between Datum ‘B’, and Datum ‘C’, but rather offset both top-to-bottom and left-to-right. The offset is shown in Figure 6 (refer to the Mechanical ICD for specific dimensions)

The common DMD datum positions and common active array center position facilitates the use of the same optical design for all DMD’s in the entire Series-450 family. The active array size and DMD Datums are defined in the DMD Mechanical ICD and illustrated in Figure 6.

3.7 **Electrical Interface Features**

The Series-450 DMD incorporates a 149-pin micro pin grid array (PGA) style of electrical interface. To achieve an electrical connection between a Series-450 DMD and the printed circuit board (PCBoard) requires a Micro-PGA socket to be installed on the system PCBoard.

The pin length, diameter, and spacing used for the Series-450 PGA are similar to that of Micro-PGA technologies used for many microprocessors, but the physical arrangement of the pins is different.

The pin numbering scheme used for Series-450 DMDs is illustrated in Figure 7.
The key features of the Series-450 electrical interface are illustrated in Figure 8 and are summarized below:

- **Corner Chamfer** – The chamfered corner of the ceramic carrier identifies the pin A1 corner as well as the intended direction of the illumination source. This provides a visual aid when installing the DMD into the socket or the DMD into the system optical interface.

- **Missing Pins** – Pins A1, A2, and B1 have been omitted to provide orientation and keying when installing the DMD into a socket.

- **Socket Seating Plane** – The socket seating plane (on the DMD) is the surface that will make contact with the DMD seating plane (on the socket). The socket seating plane is pointed out in Figure 8, and the DMD seating plane is pointed out in the cross-section shown in Figure 22.

- **Braze Fillet** – The pin is brazed to the ceramic and results in a fillet. The mating socket will need clearance to accommodate the size of the braze fillet without interfering with the proper seating of the DMD into the socket.

- **Symbolization Pad** – The symbolization pad on the pin side of the DMD ceramic carrier provides an area to mark the DMD with part number information. Note that this pad is electrically connected to signal ground.
3.8 Thermal Characteristics

The pin side of the Series-450 DMD has the thermal interface area which allows for conductive cooling of the DMD. The thermal interface includes the area of the symbolization pad and the adjacent ceramic areas, as illustrated in Figure 8. Each DMD type is characterized, during which time specific thermal test points are identified for use when verifying that the DMD temperature meets the DMD thermal specifications when installed in the end-application.

Two types of thermal specifications are provided in the DMD Data Sheet: Absolute Maximum Ratings and Recommend Operating Conditions.

Thermal specifications provided as a part of the Recommended Operating Conditions represent the temperature limits within which the DMD will meet all operational specifications.

Thermal specifications provided as part of the Absolute Maximum Ratings represent the temperature limits within which no permanent (physical) damage will occur to the DMD. Exposing the DMD to temperatures beyond the Absolute Maximum Ratings can cause irreversible damage, and should be avoided. The Absolute Maximum Ratings are provided as stress limits for use in accelerated reliability stress testing. Full-function operation of the DMD should not be expected when conditions exceed those specified in the Recommended Operating Conditions.

The temperature specifications are for specific test points identified in the data sheet and the active array. The active array temperature can not be measured directly but must be computed analytically using information in the DMD data sheet, data from a thermal test point, and the thermal load absorbed from the illumination energy. The relationship to calculate the active array temperature from this information is shown in the DMD data sheet and described in Section 3.8.2.

The image which is displayed when making the temperature measurements should be the image that produces the worst case temperatures. For an end-application where the largest thermal load is the illumination on the DMD (rather than the electrical load of the DMD) the worst case temperatures would typically result from an all black image. For an end-application where the energy on the active array is low and the thermal load on the DMD is dominated by the electrical load the worst case temperatures would typically result from a “white noise” image.
3.8.1 DLP5500 DMD (.55 XGA) Thermal Test Points

The DLP5500 DMD has two defined thermal test points identified in Figure 9. One test point is on the window side of the DMD and the other one is on the pin side. The minimum and maximum thermal requirements are summarized in Table 2, see the DMD data sheet for the table of values specific to the DMD being used. The temperatures of the reference locations can be measured directly, but the array temperature must be computed as described in Section 3.8.2.

### Table 2. Absolute Minimum and Maximum Temperatures

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<th>Parameter</th>
<th>Minimum and Maximum</th>
<th>Units</th>
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<td></td>
</tr>
<tr>
<td>• Test Point 1 in Figure 9</td>
<td>See DLP5500 data sheet</td>
<td>°C</td>
</tr>
<tr>
<td>• Test Point 2 and Array in Figure 9</td>
<td>See DLP5500 data sheet</td>
<td>°C</td>
</tr>
<tr>
<td><strong>Differential Temperature</strong></td>
<td></td>
<td></td>
</tr>
<tr>
<td>• Test Point 1 in Figure 9 minus Array</td>
<td>See DLP5500 data sheet</td>
<td>°C</td>
</tr>
<tr>
<td><strong>Storage Temperature (non-operating):</strong></td>
<td></td>
<td></td>
</tr>
<tr>
<td>• Test Point 1 and Test Point 2 in Figure 9</td>
<td>See DLP5500 data sheet</td>
<td>°C</td>
</tr>
</tbody>
</table>

![Figure 9. Thermal Test Points](image-url)
3.8.2 Array Temperature and its Calculation

The total thermal load on the DMD is a result from the electrical power dissipated by the DMD, plus the optical energy absorbed by the DMD. The electrical load to be used for the active array calculations should be measured when possible. If measurement is not possible a typical value (associated with a display application) is identified in the DMD data sheet. The energy absorbed from the illumination source is variable and depends on the operating state of the mirrors, the intensity of the light source, and the distribution of overfill illumination. The energy absorbed from the optical load must be determined for each specific end-application and each specific illumination design.

The array temperature can be calculated using the formulas below.

\[
T_{\text{Array}} = T_{\text{Ceramic}} + (Q_{\text{Array}} \times R_{\text{Array - To - Ceramic}})
\]

\[
Q_{\text{Array}} = Q_{\text{ELE}} + Q_{\text{ILL}}
\]

Where:

- \(T_{\text{Array}}\) = computed active array temperature (°C)
- \(T_{\text{Ceramic}}\) = temperature measured at the thermal test point on the pin side (°C)
- \(Q_{\text{Array}}\) = total DMD active array thermal load (electrical + absorbed) (W)
- \(R_{\text{Array - To - Ceramic}}\) = thermal resistance between the thermal test point and the DMD active array (°C/W) (specified in the DMD data sheet)
- \(Q_{\text{ELE}}\) = electrical power consumption (W)
- \(Q_{\text{ILL}}\) = absorbed optical energy (W) (end-application specific)

The DMD thermal load can vary from unit-to-unit due to variations in the illumination source. Therefore, when verifying the thermal design of a specific end-application it is important to verify the amount of illumination energy each time a temperature measurement is taken.

The thermal specifications provided in the DMD data sheets are based upon characterizations done with illumination loads which are evenly distributed across the active array with less than 16% (by energy) of overfill. Applications utilizing illumination profiles which have regions of high energy density (for example, highly collimated laser beams) have not been characterized and require special consideration on the part of the product designer.

The primary thermal load on the DMD originates from the dissipated electrical load plus the absorbed optical load. The primary thermal dissipation path for the DMD is by conduction through the thermal interface area defined on the pin side of the Ceramic Carrier. Secondary heating and cooling paths can exist, but their significance depends upon the magnitude and location of any associated (secondary) thermal loads.

The thermal load on the active array has a low resistance direct conduction path to the thermal interface area on the pin side of the Ceramic Carrier. The primary thermal dissipation path for the energy on the window aperture is the same thermal interface area on the pin side of the Ceramic Carrier. The conduction path from the window aperture to the thermal interface area is much higher than for the active array. Therefore, the thermal load on the window aperture should be minimized (none is best).

Note that optical energy that falls on the window aperture must be cooled but does not contribute to the optical efficiency of the DMD.
3.8.2.1 Sample DLP5500 Active Array Calculation for a 1-Chip Display Application

For a typical 1-chip display application the thermal load on the DLP5500 DMD from the illumination has been characterized to a factor based on average measured screen intensity. A conversion factor for energy on the DMD based on measured screen intensity (lumens) is 0.00288 times the measured screen lumens. This is based on the following:

- efficiency from DMD to the screen of 87%
- spectral efficiency of 300 lumens/watt of projected light
- illumination distribution on the DMD of
  - 83.7% on the active array
  - 16.3% on the active array border and window aperture

An example of the active array temperature calculation for this display application using a DLP5500 (.55 XGA Series-450) DMD is shown below.

\[
Q_{\text{Array}} = Q_{\text{ELE}} + Q_{\text{ILL}}
\]

\[
Q_{\text{ILL}} = 0.00288 \times S_l
\]

Where:

- \(T_{\text{Array}}\) = computed active array temperature (°C)
- \(T_{\text{Ceramic}}\) = temperature measured at thermal test point TP2 (°C)
- \(Q_{\text{Array}}\) = total DMD active array thermal load (electrical + absorbed) (W)
- \(R_{\text{Array - To - Ceramic}}\) = thermal resistance between the thermal test point 2 and the DMD active array is 0.6°C/W (from DMD data sheet)
- \(Q_{\text{ELE}}\) = electrical power consumption is 2.0 W (from DMD data sheet)
- \(S_l\) = measured screen intensity (lumens)
- \(Q_{\text{ILL}}\) = absorbed optical energy for this display application (W)

Then:

\[
Q_{\text{Array}} = 2.0 + (0.00288 \times 2000) = 7.76 \text{ W}
\]

\[
T_{\text{Array}} = 55^\circ C + (7.76 \text{ W} \times 0.6^\circ C/W) = 59.7^\circ C
\]

3.8.3 Temperature and UV

In addition to specifying the Absolute Maximum and Recommended Operating temperature ranges, the DMD data sheet specifies the maximum UV power density which should be incident upon the active array and/or overfill areas. To ensure the longest possible reliability the maximum operating temperature and maximum UV levels should not occur at the same time.
3.9  **Mechanical Loading**

Installing a DMD into an end-application environment will involve placing a mechanical load on the DMD, and (more specifically) upon the Ceramic Carrier. The maximum mechanical load which can be applied to the DMD is specified in the DMD Data Sheet. The areas the loads are to be distributed are shown in Figure 10. The maximum loads apply for both the installation process and the continuous load after the DMD has been installed.

**Thermal Interface Area**

The Series-450 DMD is designed to accommodate a mechanical load evenly distributed across the thermal interface area shown in Figure 10. This load is required to interface a heat sink (or other thermal hardware) in order to facilitate optimal thermal performance. The minimum mechanical load applied to this area is that which is needed to ensure good conductivity of the thermal pad. A graph indicating thermal conductivity and pressure is generally available from the thermal pad manufacturers. When determining the mechanical load applied to the Thermal Interface area the manufacturing tolerances of the parts as well as worst-case mechanical shock loads expected should be considered.

**Electrical Interface Area**

The Series-450 DMD is designed to accommodate mechanical loads evenly distributed across each of the electrical interface areas shown in Figure 10. The Micro-PGA sockets designed for use with the Series-450 DMDs do not require a continuous load after the DMD is installed in the socket. The load applied to the electrical interface areas generally results from mounting the DMD PCBoard. The only mechanical load required on these areas is that which is required to support the mass of the DMD, Micro-PGA socket and PCBoard under mechanical vibration and shock conditions.

**Datum ‘A’ Areas**

The Series-450 DMD will accommodate a mechanical load evenly distributed across the three Datum ‘A’ areas shown in Figure 10. This load functions to counteract the combined loads from the thermal and the electrical interface areas. The Mechanical ICD defines the location and size of the Datum ‘A’ areas.

Loads in excess of the specified limits can result in mechanical failure of the DMD package.

![Figure 10. DMD Mechanical Loads](image-url)
4 System DMD Mounting

4.1 Critical Considerations for Mounting the DMD

The method used to mount the DMD into the end-application system needs to meet the functional design objectives of the application, while also insuring that the DMD thermal and mechanical specifications are not exceeded.

The functional design objectives of the mounting system include:

• to establish (and maintain) the physical placement of the DMD’s active array relative to the optical axis of the applications optical assembly
• to establish (and maintain) a dust-proof seal between the DMD and the chassis of the optical assembly
• to establish (and maintain) a reliable electrical connection between the DMD’s electrical interface and the system socket
• to establish (and maintain) a proper thermal connection between the DMD’s thermal interface area and the system’s thermal solution

To meet these functional design objects requires that some minimum mechanical load be applied to the DMD. The DMD mounting concepts presented in this application note achieve the minimum mechanical load to meet the functional objectives while illustrating various concepts for controlling the maximum mechanical loads being applied to the DMD.

The ideal design is one which:

• does not rely upon strict assembly techniques or processes (see Section 4.2.6.2)
• is tolerant of manufacturing variations of piece parts
• minimizes the variations in mechanical loads applied to the DMD

If not understood and minimized the variations can easily result in lower forces than what is needed for good thermal contact or higher forces which could result in damage to the DMD.

4.2 Basic System DMD Mounting Concept

The DMD mounting concepts described in this application note represent “drop-in-place” designs. The “drop-in-place” name indicates that the DMD is placed onto the optical chassis mounting features and secured into place without any adjustment for the optical alignment of the DMD. A “drop-in-place” design is desirable because it simplifies the assembly process of the DMD into the system. Achieving a “drop-in-place” design is realistic for a single-chip DMD system. Achieving a “drop-in-place” design for a multi-DMD system is more challenging, due to the need to align the individual DMD’s to each other in order to form a single combined image.

When using “drop-in-place” mounting concepts the illumination light bundle still needs to be aligned to the DMD active array. Generally the illumination light bundle is adjusted to align it to the DMD after the DMD is installed into the system. A convenient way to perform this adjustment is by adjusting an integrator element or fold mirror.

The “drop-in-place” style of mounting simplifies the assembly of the DMD into the optical assembly, but adequate tolerances are required on the DMD interface features of the optical chassis (see Section 4.2.1). The specific tolerance requirements vary for each system design. Key areas of consideration include:

• alignment of the illumination light bundle to the active array (X-axis, Y-axis, and rotation)
• size and location of the illumination overfill
• focus across the entire active array
• variation in the location (and rotation) of the active array relative to the illumination light bundle due to size and location tolerances of the DMD mounting features (optical interface) on the optical chassis (this is less critical if DMD interchangeability is not important)
• variation in the location (and rotation) of the active array within the DMD package due to size and location tolerances of the DMD datum features, and the placement of the active array relative to the datum features (this is less critical if interchangeability of DMDs is not important)

Alignment of the illumination light bundle to the active array, the overfill size, and light bundle location are interrelated. The illumination alignment range needs to comprehed the overfill size and dimensional

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tolerance of the piece parts. Adjustment of the illumination is required unless an excessive amount of overfill is used. Note however that excessive overfill increases the amount of DMD cooling required and reduces the efficiency of the system (both optical efficiency and electrical power efficiency). For these considerations it is nearly always best to minimize the amount of overfill (size) and to design the system and process with alignment in mind.

A key characteristic of the “drop-in-place” mounting concept is that the planarity of the DMD does not need to be adjusted in order to achieve acceptable focus across the entire active array. The depth-of-focus of the optical design is critical to achieving acceptable focus. Key considerations when determining the depth-of-focus required by the optical design include:

- the angular relationship between the DMD Datum ‘A’ mounting points, the corresponding Datum ‘A’ points on the optical chassis, and the features used to mount the projection lens (optical axis) to the optical chassis. (Typically this translates to a parallelism or perpendicularity between the indicated surfaces depending on the specific optical design)
- parallelism of the DMD active array to the three Datum ‘A’ areas

One basic approach for a “drop-in-place” mounting of a Series-450 DMD is illustrated in Figure 11 and described below:

- Interface, optical – contains features which are the corresponding features of the DMDs Datum ‘A’, Datum ‘B’ and Datum ‘C’ features. These features are described in more detail in Section 4.2.1. The part, as shown, is not intended to be used as a standalone part but rather as a reference for the features to be incorporated into the system optical chassis. Additional features can be included into this part which aid or assist the installation and mounting of the DMD and DMD PCBoard (see Section 4.3.1 and Section 4.3.2 for information regarding the features).
- System Aperture – located between the DMD and optical chassis. The aperture functions to prevent or reduce stray and scattered light from becoming part of the projected image.
- Dust Gasket (dust seal) - functions to provide a basic environmental barrier between the system optical chassis and DMD, specifically to prevent ambient dust particles from reaching the DMD window glass.
- Series-450 DMD – any one of the DMDs which constitute the family of Series-450 DMDs
- s450 Socket – functions to provide the electrical connection between the DMD and the applications DMD PCBoard. Section 5 contains further information about the socket designs.
- DMD PCBoard – the electronics board the DMD is mounted on
- Thermal Pad – functions to maximize the transfer of heat from the DMD (thermal contact area) to the system thermal solution (heat sink). Be aware that the DMDs Thermal Interface area is electrically connected to signal ground (through the Ceramic Carrier), and that some thermal pad materials are electrically conductive.
- Heat Sink – functions to conduct heat from the DMD to a larger (external) surface area where the heat can be transferred to the ambient environment surrounding the product
- Compression Springs - function to absorb the manufacturing tolerances for the parts used to mount the DMD heat sink to the optical chassis. The spring selection (or design) is critical for control of both the minimum and maximum mechanical loads on the DMD. See Section 4.2.2 for more details.
- Shoulder Screws – function to mount the heat sink to the optical chassis. Along with the compression springs the shoulder screw design is critical for controlling the minimum and maximum mechanical loads applied to the DMD thermal area. See Section 4.2.2 for more details.
- Push Nut – functions as a retainer, making the shoulder screws and compression spring captive to the heat sink.
- Heat Sink Assembly – comprised of the heat sink, thermal pad, shoulder screws, compression springs, and push nuts. Makes these parts a stand-alone sub-assembly, thus simplifying the process of installing a DMD into a product.
Not shown in Figure 11 but equally important is the method used to mount the DMD PCB board assembly into the product. The specific method used to secure the PCB board into the product can result in a wide range of loads applied to the DMD electrical interface areas. This is discussed more in Section 4.2.6.

4.2.1 Optical-Mechanical Alignment Features

The DMD Optical-Mechanical Alignment Features (datums) are used to establish and maintain the physical placement of the DMD’s active array relative to the illumination light bundle and the optical axis of the projection lens. Earlier sections reviewed the Optical Interface Features of the DMD. This section reviews the suggested corresponding features on the optical chassis. The features shown in Figure 12 are summarized below:

- Datum ‘A’ tabs - three coplanar areas that contact the DMD Datum ‘A’ areas
- Datum ‘B’ round pin – mates with the DMD Datum ‘B’ hole
- Datum ‘C’ pin – mates with the DMD Datum ‘C’ slot
- Two threaded bosses to mount a heat sink, and secure the DMD against the Datum ‘A’ features of the system optical chassis.

These features on the optical chassis are commonly referred to as the optical interface.
The following characteristics of the Series-450 Optical-Mechanical alignment features should be noted:

- The different size and shapes of the DMD’s Datum ‘B’ and Datum ‘C’ features (hole and slot) naturally provides for physical keying of the DMD when being installed into the optical chassis.
- The three Datum ‘A’ tabs on the optical chassis must be coplanar to ensure uniform focus of the active array, and focus repeatability between systems. The coplanarity of these features and the DMD parallelism combine to determine the requirements for the depth of focus for the optical system.
- The outline shape of the features on the optical chassis that contact the DMD Datum ‘A’ features should be smaller than the defined DMD Datum ‘A’ features to ensure the area outside the DMD Datum ‘A’ area is not contacted. Contact outside of the DMD Datum ‘A’ area could result in focus variations or non-uniform focus.
- The Datum ‘B’ and ‘C’ features on the DMD (hole and slot) are not the full depth of the Ceramic Carrier. For this reason the maximum length of the mating features (pins) on the optical chassis must be controlled to keep the top of the pin from contacting the bottom of the DMD Datum holes before the Datum ‘A’ surfaces have been properly contacted.
- The system gasket or aperture (if used) should be designed to not interfere with the proper mating of the DMD Datums and corresponding Datum ‘A’ features on the optical chassis. Any gasket or aperture material which overlaps the DMD Datum ‘A’ features could cause focus problems. Another issue that could result in focus problems is if the gasket material is not compliant enough to allow sufficient compression, thus prohibiting full contact of all the Datum ‘A’ features.
- The base of the Datum ‘B’ and Datum ‘C’ pins on the optical chassis could be used to align a system aperture or dust gasket. See Section 4.2.5.
- Sharp edges on the Datum ‘A’ tab features should be avoided in order to prevent damage to the DMD Ceramic Carrier. The sharp contact point of a feature edge could result in a highly concentrated load in a very small area, potentially leading to cracking the DMD’s Ceramic Carrier.

4.2.2 Heat Sink Mounting

The system thermal solution (heat sink) is intended to contact the DMD Thermal Interface area. The method used to mount the heat sink to the optical chassis will determine the mechanical load being applied to the DMD Thermal Interface area. The minimum required load will depend on the thermal pad used. The maximum load which should be applied to the DMD is described in Section 3.9.

One of the key features of the mounting concept shown here is that the minimum and maximum
mechanical loads on the DMD can be controlled by design, and does not rely on assembly processes (techniques). The loads are controlled through design and the use of compression springs in conjunction with shoulder screws. The springs can be selected (or designed) such that the forces applied to the DMD thermal area do not exceed the DMD specification, but yet provide sufficient force to ensure good thermal conductivity of the thermal pad.

The first step in this type of design is to understand the size and variation of the gap into which the spring will fit. The variations in gap, along with the variations in spring size, and spring-rate will determine the force variation applied to the DMD Thermal Interface area. The minimum force occurs when the parts manufacturing tolerances result in the largest gap, while the maximum forces occur when the parts manufacturing tolerances result in the smallest gap.

For the design concept illustrated in Figure 11 the parts which contribute to the spring gap variation are the shoulder screw, the optical chassis (specifically the DMD interface features), the DMD, the thermal pad, and the heat sink stud. The gap the spring will fit (called 'critical gap') is shown in Figure 13. Refer to Section 4.3.1.1 for a detailed example.

![Figure 13. Heat Sink Mounting Critical Gap](image)

### 4.2.3 Dust Gasket

The dust gasket provides a barrier to prevent ambient dust particles from accumulating on the DMD window glass. The outside window surface is relatively near the image plane (active array) of the DMD. The cross section view of the DMD shown in Figure 4 illustrates the close proximity.

Dust particles on the DMD window, if large enough, could appear in the projected image as shadows or near shadows.

Characteristics of a dust gasket should include:
- creates no interference with the DMD mounting features (Datum ‘A’, ‘B’, and ‘C’) on the optical chassis when in either the compressed or non-compressed state
- has sufficient compliance to allow necessary compression without a significant mechanical mounting load on the DMD
- creates a sufficient seal against the surfaces it contacts to prevent dust particles from reaching the DMD window glass
- comprised of a material which does not create particles
- comprised of a material which does not allow dust particles to pass through it’s volume

### 4.2.4 System Aperture

The system aperture functions to absorb or reflect stray light (either reflected or scattered) from entering the projection lens pupil and thus becoming part of the projected image. The source of the stray light could be from the surface of optical components near the DMD (typically projection lens elements), mechanical mounting parts for the optical components, or areas of the DMD outside the active array.
Characteristics of an aperture should include:

- a suitable surface finish (one which will minimize light scattering)
- a suitable surface color (generally an absorptive flat black produces the best results)
- an aperture opening edge which minimizes light scattering (sharp rather than rounded, smooth free of nicks and irregularities, etc…)
- shape and dimensions to complement the overall optical design (illumination f-number; characteristics of optical elements near the active array like shape, curvature, distance, location, etc…)
- a suitable surface angle orientation (tilting) relative to the active array plane

It is expected that the size and shape of the opening in any TI reference aperture design would require modification, depending on the specific details of a products optical design.

The usual method to determine the final aperture design (opening size, shape, and tilt relative to the active array plane) is to iterate the aperture design features once it is possible to project an actual image. This way iterations can be performed until any objectionable scattered light has been eliminated or reduced to a satisfactory level.

4.2.5 System Aperture and Dust Gasket Mounting

The system aperture and dust gasket can be installed as separate parts or attached together before installation. The concept illustrated in Figure 11 has separate parts that are compressed between the DMD and the optical chassis.

The Datum ‘B’ and ‘C’ features on the DMD are small in diameter and have limited depth. This makes the corresponding features on the optical chassis difficult to use for the X and Y-axis alignment of an aperture or gasket. The datum features ‘A’, ‘B’, and ‘C’ on the optical chassis can be raised above a flat surface as shown in Figure 14. If the size and tolerance of these features are properly controlled the base of these features can be used to align the aperture and gasket.

Note that using the base of these features as described may require an additional recess (around the bottom of the feature) to accommodate a fillet which is typically present. This is illustrated in Figure 14.

It is desirable to have the system aperture located at or very near the outside surface of the DMD window glass as shown in Figure 14. This is complicated by the DMD window glass protruding from the Datum ‘A’ surface making it difficult to support simple flat style apertures against the area of the DMD adjacent to the Datum ‘A’ areas. Alternatively, the system aperture can be located against the flat surface of the optical chassis as shown in Figure 14. The height of the protruding features on the optical chassis need to be controlled sufficiently to ensure the aperture is the desired distance from the DMD window. The aperture can be held against the optical chassis by the compressive forces of the dust gasket as shown in Figure 14.

Figure 14. System Aperture and Gasket
Another option for the location of the dust gasket is illustrated in Figure 15. In this approach the dust gasket is compressed between the DMD PCBoard and the optical chassis, rather than between the DMD and optical chassis. This places the dust gasket around the perimeter of the DMD and Socket. This requires a thicker gasket but the compressibility is less sensitive to thickness variations of the gasket material, or the gap into which it is compressed.

![Figure 15. Gasket Location Option](image)

### 4.2.6 DMD PCBoard Mounting

The DMD is primarily held in place (on the optical chassis) by the heat sink assembly and does not require any additional securing. However, the DMD PCBoard must be mounted in a manner which prevents it from moving during mechanical shock, mechanical vibration, or inadvertent movement during a subsequent assembly operation (like attaching another electronics assembly). The PCBoard typically requires only a minimum amount of securing to prevent disengagement of the DMD from the socket (although, this does depend on the size of the PCBoard, because the larger the PCBoard the larger the mass and moment arm).

The method used to mount the PCBoard is critical for controlling the load applied to the DMD’s electrical areas. The maximum allowable load for the DMDs electrical areas is specified in the DMD Data Sheet and described in Section 3.9.

Considerations for mounting the DMD PCBoard include:

- prevention of any movement which may result in separation of the DMD Datum ‘A’ features and the corresponding features on the optical chassis
- prevention of any tilting of the DMD about the optical chassis Datum ‘A’ surface
- prevention of any flexing or bending of the DMD PCBoard which could result in damage to the solder joints of the socket or other components on the PCBoard
- prevention of any movement of the DMD caused during the assembly of other electronic assemblies
- minimizing the size and weight of the PCBoard (if important to the end product)
- simplification of assembly processes and preferred assembly methods

The DMD PCBoard can be mounted in many ways. The optimum mounting method in a specific end product will dependant upon the geometry and design objectives of the product. The many possible mounting methods can be categorized as either "direct control" or "indirect control", according to how the resulting mechanical forces on the DMD are controlled.

#### 4.2.6.1 Direct Control

A variety of direct control mounting techniques for the DMD PCBoard are illustrated in Figure 16. These techniques provide a direct means of controlling the forces applied to the DMD. Additionally, the force applied to the DMD by these techniques can often be negligible.
The "Heat Sink Bracket to PCBoard" or "Interface Bracket to PCBoard" techniques apply no forces to the DMD. Of these techniques the "Interface Bracket to PCBoard" option is the simplest to design and implement.

A "direct control" method has fewer critical piece-part tolerances, is less dependant upon the assembly processes, is believed to result in fewer assembly issues, and results in fewer (no) DMDs being damaged. The three options included in this application note are intended to stimulate ideas for implementing a "direct control" mounting method. The specific details can vary widely. Some of the pro's and con's for these options include:

- "Interface Bracket to PCBoard" mounting option
  + simplest and most direct method to design and implement
  + results in no forces added to the heat sink springs
  + results in no movement of PCBoard
    – can not be assembled top down (when installing screws)
    + slot allows adjustment of bracket (to accommodate part tolerances)

- "Heat Sink Bracket to PCBoard" mounting option
  + simple and direct method to design and implement
  – force applied to the heat sink springs when preventing PCBoard movement
  – reduction in force applied to the heat sink when PCBoard movement occurs
  – small amounts of movement may be possible in high shock environments
  – can not be assembled top down (when installing screws)

---

**Figure 16. DMD PCBoard Direct Control Mounting Techniques**

The "Heat Sink Bracket to PCBoard" or "Interface Bracket to PCBoard" techniques apply no forces to the DMD. Of these techniques the "Interface Bracket to PCBoard" option is the simplest to design and implement.

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- "Interface Bracket to PCBoard" mounting option
  + simplest and most direct method to design and implement
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  + results in no movement of PCBoard
    – can not be assembled top down (when installing screws)
    + slot allows adjustment of bracket (to accommodate part tolerances)

- "Heat Sink Bracket to PCBoard" mounting option
  + simple and direct method to design and implement
  – force applied to the heat sink springs when preventing PCBoard movement
  – reduction in force applied to the heat sink when PCBoard movement occurs
  – small amounts of movement may be possible in high shock environments
  – can not be assembled top down (when installing screws)
+ slot allows adjustment of bracket (to accommodate part tolerances)

• "Compression Washer Near Edge" mounting option
  + top down assembly method is possible (when installing screws)
  + shoulder screws and washer compliance are key to controlling forces on DMD
  – variation in washer material compression will cause variation in forces
  – difficult to get or develop force vs. deflection data for the washer material; this makes it difficult to
calculate the force characteristics
  – part tolerances (shoulder screw, washer, interface boss for screw) are more critical than for a
simple bracket

4.2.6.2 Indirect Control

Using screws to clamp the PCBoard against bosses on the optical chassis (as shown in Figure 17 can
easily result in large mechanical forces being applied to the DMD. In this situation there are multiple items
which indirectly apply force to the DMD. Therefore, controlling the force on the DMD requires close
attention to the force contributed by each item. This is a difficult analysis to do because the assembly
process is typically a significant factor.

With this mounting method, the items that influence the forces on the DMD (and thus contribute to the
variation and unpredictability of the forces) include:
• size of gap between boss on the optical chassis and the PCBoard
• PCBoard stiffness (affected by layer count, material properties, thickness, etc…)
• diameter of the screws
• friction factors between the screws and the optical chassis threads
• amount of torque applied to the screws
• method of tightening the screws (order, partial tightening all screws before final tightening of any
  screw, etc…)
• distance between the mounting screws and the DMD

Minimizing the ‘critical gap’ (shown in Figure 17 ) between the boss on the optical chassis and the
PCBoard minimizes PCBoard bending. In general the larger the ‘critical gap’ the more potential for
PCBoard bending, and the larger the resultant forces on the DMD.

Understanding the minimum and maximum size of the ‘critical gap’ can be done by tolerance analysis of
the parts and features that contribute to the gap. The maximum size gap is an indication of how much the
PCBoard may bend, and the minimum size gap indicates if the DMD Datum ‘A’ features will make contact
with the corresponding features on the optical chassis. If the DMD Datum ‘A’ features are not in full
contact with the corresponding features on the optical chassis this could impact the focus and focus
uniformity of the DMD. Refer to Section 4.3.1.2 for a detailed example.

It is difficult (nearly impossible) to keep the PCBoard from bending by use of torque on the mounting
screws, and still have the screws sufficiently tight to prevent the screws from becoming loose during
vibration conditions. In this type of situation, thread locking materials may be required to prevent the
screws from loosening.

Figure 17. DMD PCBoard Mounting Gap
4.3  **DMD Mounting Concept Details**

Detailed concepts for mounting the DMD are described in this section. The concepts are similar except for the features which aid the installation of the DMD, and the related parts which are needed to accommodate the different interface features. “Edge Guide” (see Section 4.3.1) and “Pin Guide” (see Section 4.3.2) mounting concepts are described in this document.

It is expected that the parts and features represented in the TI concept designs will be adapted or modified by the customer for their specific application, part design requirements, part manufacture requirements, and other customer needs. The surface area of the heat sink in the concept design is not intended to be representative of the surface area needed for adequate cooling of the DMD. The heat sink illustrates the features necessary for mounting the DMD and heat sink. The actual heat sink design will need to account for the amount of cooling air available, ambient air temperature, thermal load on the DMD, and other characteristics of a particular system thermal design.

4.3.1  **Edge Guide Mounting Concept**

The design concept for mounting the Series-450 DMD shown in Figure 18 incorporates specific features on the optical chassis intended to aid DMD alignment during the DMD installation process. While the corresponding features on the optical chassis and those on the DMD (Datum ‘A’, ‘Datum B’, and Datum ‘C’) could be used as the sole alignment aids, these features are small and generally can not be seen during the assembly process. The rough alignment provided by the protruding edge guide features are thought to simplify assembly, and reduce the chance of damaging the DMD window due to contact with the Datum ‘B’ and Datum ‘C’ alignment pins (on the optical chassis).

The edge guide features shown in Figure 18 are intended to provide an initial rough alignment to the edges of the DMD ceramic carrier. The Datum ‘A’, ‘B’, and ‘C’ features (described in earlier sections) are intended to provide the final alignment. The detailed analysis of the critical gaps associated with mounting the heat sink is covered in Section 4.3.1.1. The details of securing the PCBoard by attaching it to the optical chassis is covered in Section 4.3.1.2.

The drawing number for the “Edge Guide” mounting concept assembly shown in Figure 18 is 2509652. The 3D-CAD models (in STEP format) and drawings (in pdf format) for each of the parts shown in Figure 18 are available for download. See Section 6.

![Figure 18. DMD Mounting Concept – Edge Guide](image)
4.3.1.1 Heat Sink Mounting

The method of mounting the heat sink shown in Figure 18 uses shoulder screws and compression springs to control the forces on the DMD Thermal Interface area. The compression springs are compressed between the head of the shoulder screw and the heat sink. The variation in this gap (shown as 'critical gap' in Figure 19) is the result of the part manufacturing tolerances. Figure 19 illustrates the key part features and a schematic of the tolerances. The tolerance schematic starts at the head of the shoulder screw (on the left-hand side of the figure), and continues around to the surface of the heat sink that contacts the spring (on the right-hand side of the figure). The identified 'critical gap' is the difference between the “start” and “end” locations.

![Figure 19. Heat Sink Mounting Tolerance Analysis Schematic](image)

Table 3 lists the nominal value and tolerance for each feature that contributes to the size of the 'critical gap'. The nominal values and tolerances used are those on the part drawings for the “Edge Guide” mounting concept shown in Figure 18. The parts are dimensioned in a manner to minimize variations in the 'critical gap'. The tolerance analysis indicates the nominal gap size is 7.1294 mm. The simple sum (SUM) method of tolerance analysis (worst case) yields a tolerance of ±0.6600 mm which results in a minimum gap of 6.469 mm, and a maximum gap of 7.789 mm. The root sum square (RSS) method of tolerance analysis yields a tolerance of ±0.3302 mm which results in a minimum gap of 6.799 mm and a maximum gap of 7.460 mm.

<table>
<thead>
<tr>
<th></th>
<th>Nominal (mm)</th>
<th>Direction Sign</th>
<th>Nominal (mm)</th>
<th>Tol (±) (mm)</th>
</tr>
</thead>
<tbody>
<tr>
<td>Shoulder screw</td>
<td>15.8100</td>
<td>−1</td>
<td>−15.8100</td>
<td>0.0800</td>
</tr>
<tr>
<td>Interface, optical (Datum 'A' to shoulder screw boss)</td>
<td>4.5800</td>
<td>−1</td>
<td>−4.5800</td>
<td>0.1600</td>
</tr>
<tr>
<td>DMD</td>
<td>2.9500</td>
<td>1</td>
<td>2.9500</td>
<td>0.2400</td>
</tr>
<tr>
<td>Thermal pad (compressed height)</td>
<td>0.4826</td>
<td>1</td>
<td>0.4826</td>
<td>0.0500</td>
</tr>
<tr>
<td>Heat sink</td>
<td>9.8280</td>
<td>1</td>
<td>9.8280</td>
<td>0.1300</td>
</tr>
<tr>
<td>SUM</td>
<td>–7.1294 (1)</td>
<td></td>
<td>0.6600 SUM</td>
<td>6.469</td>
</tr>
<tr>
<td>RSS</td>
<td></td>
<td></td>
<td>0.3302 RSS</td>
<td>6.799</td>
</tr>
</tbody>
</table>

(1) Negative indicates a gap for the spring
The resultant force on the DMD Thermal Interface area depends upon:
- the minimum and maximum 'critical gap' size for the compression spring (from above analysis)
- the nominal free length of the spring
- the tolerance of the spring free length
- the nominal spring rate
- the tolerance of the spring rate

For the design concept illustrated in Figure 18 the minimum force required for thermal performance of the thermal pad is determined as follows:
- area of the heat sink stud and thermal pad is $12.0 \times 18.0 = 216 \text{ mm}^2$
- to achieve a thermal pad resistance of 0.58°C/W will require a pressure of 185 KPa on the thermal area (pressure from thermal pad data sheet)
- to achieve a pressure of 185 KPa on the 216 mm² thermal pad area would require a total load of 40 Newton be applied to the DMD Thermal Interface area
- since two compression springs are used the minimum needed per spring is 20 Newton

For a maximum load of 111 Newton applied to the DMD Thermal Interface area, the load per spring would be 55.5 Newton.

Thus, the compression spring to be selected (or designed) for this application should produce a minimum of 20 Newton and a maximum of 55.5 Newton. For springs reviewed in this force range the typical spring free length tolerance was ±0.56 mm, and the spring rate tolerance was ±10% of the nominal spring rate.

The nominal force occurs when:
- the gap is 'as designed' (no tolerances associated with part manufacturing applied)
- the spring rate is the nominal specified value
- the free length is the 'as designed' length (no tolerances applied)

The maximum force occurs when:
- the gap is the smallest (using either SUM or RSS tolerance analysis)
- spring rate is at the high side of tolerances
- free length of the spring is on the high side of the tolerances

The minimum force occurs when:
- the gap is the largest (using either SUM or RSS tolerance analysis)
- spring rate is on the low side of tolerance
- free length of the spring is on the low side of the tolerances

Table 4 summarizes the SUM (worst case) and RSS tolerance analysis results for several candidate springs.

<table>
<thead>
<tr>
<th>Spring</th>
<th>Outside Diameter (mm)</th>
<th>Wire Diameter (mm)</th>
<th>Solid Length (mm)</th>
<th>Rod Diameter (mm)</th>
<th>Free Length (mm)</th>
<th>Spring Rate (N/mm)</th>
<th>Worst Case (SUM)</th>
<th>RSS Case Tolerance</th>
</tr>
</thead>
<tbody>
<tr>
<td>A</td>
<td>7.619</td>
<td>0.965</td>
<td>5.105</td>
<td>5.460</td>
<td>11.125</td>
<td>7.72</td>
<td>30.85</td>
<td>19.29 44.28</td>
</tr>
<tr>
<td>B</td>
<td>7.619</td>
<td>1.016</td>
<td>5.537</td>
<td>5.359</td>
<td>11.125</td>
<td>9.91</td>
<td>39.60</td>
<td>24.77 56.84</td>
</tr>
<tr>
<td>C</td>
<td>8.991</td>
<td>1.092</td>
<td>5.054</td>
<td>6.578</td>
<td>10.998</td>
<td>9.87</td>
<td>38.18</td>
<td>23.54 55.23</td>
</tr>
<tr>
<td>D</td>
<td>9.143</td>
<td>1.066</td>
<td>5.105</td>
<td>6.781</td>
<td>11.125</td>
<td>8.15</td>
<td>32.56</td>
<td>20.37 46.75</td>
</tr>
</tbody>
</table>

The results of the tolerance analysis indicates:
- all springs (A, B, C, D) have a nominal force that is between the minimum and maximum loads required (20.0 - 55.5 N).
- spring (A) has a worst case (SUM) load for minimum tolerances (19.29 N) just under the 20.0 N minimum, but the RSS load (21.58 N) is greater than the minimum needed.
- spring (B) has a worst case (SUM) load for the maximum tolerances (56.84 N) over the 55.5 N maximum, but the RSS load (53.25 N) is less than the maximum requirement.
• spring (C) has loads that are between the minimum and maximum loads required although the worst case (SUM) load (55.23 N) is just below the maximum requirement
• spring (D) has loads that are between the minimum and maximum loads required although the worst case (SUM) load (20.37 N) is just above the minimum required

A final selection between these spring candidates would be based on several factors, including but not limited to:
• what the likely hood that the worst case (SUM) conditions would occur?
• if the worst case (SUM) conditions did occur what type of failure is likely to result? (Would the DMD be damaged? Would the electrical connection to the DMD be compromised?, etc…)
• if a failure did result, could it be identified before the product was shipped to the final customer?
• if a failure did result, what would the impact on the final customer be?

4.3.1.2 PCBoard Mounting

It is recommended that the PCBoard be mounted using some type of a “direct control” method, like those described in section Section 4.2.6.1. A method based on the ‘Interface Bracket to PCBoard’ is the simplest. The details of the bracket designs are very specific to the geometry of the end product and will not be covered in detail.

Analysis of the ‘critical gap’ between the PCBoard and the boss on the optical interface shown in Figure 20 is not needed when a direct control mounting method is used, but such an analysis may help to understand the potential for PCBoard bending when making a decision on which mounting method to use.

One of the key items to implement the "indirect control" method (if chosen) is to minimize bending of the DMD PCBoard. Minimizing the 'critical gap' (shown in Figure 20) between the DMD PCBoard and the corresponding boss on the optical chassis will reduced the potential for bending.

This is achieved by minimizing the gap between the DMD PCBoard and the corresponding boss on the optical chassis (optical interface). The parts in Figure 18 which contribute to the 'critical gap' are shown in Figure 20 along with a tolerance schematic. The tolerance schematic starts at the top of the mounting screw boss on the optical chassis (in the top left-hand corner of the figure) and continues around to the face of the PCBoard closest to the optical chassis (in the top right-hand corner of the figure). In the ideal situation the PCBoard would be infinitely rigid and would not bend due to the force applied by the screw, but this is not the case. The PCBoard is likely to bend and the largest size of the 'critical gap' determines the maximum amount of bending that can occur.

![Figure 20. PC Board Gap Analysis Schematic](image)

Table 5 lists the nominal value and tolerance for each feature that contributes to the size of the 'critical gap'. The nominal values and tolerances used are those on the part drawings for the "Edge Guide" mounting concept shown in Figure 18. The parts are dimensioned in a manor to minimize variations in the
“critical gap”. The tolerance analysis indicates the nominal gap size is 0.6500 mm. The simple sum (SUM) method of tolerance analysis (worst case) yields a tolerance of ±0.6400 mm which results in a minimum gap of 0.010 mm, and a maximum gap of 1.290 mm. The root sum square (RSS) method of tolerance analysis yields a tolerance of ±0.3709 mm which results in a minimum gap of 0.279 mm and a maximum gap of 1.021 mm.

### Table 5. Gap Between Interface Boss and PCBoard

<table>
<thead>
<tr>
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<th>Nominal (mm)</th>
<th>Direction Sign</th>
<th>Nominal (mm)</th>
<th>Tol (±) (mm)</th>
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<tr>
<td>Interface, optical (Datum ‘A’ to PCBoard boss)</td>
<td>5.0000</td>
<td>−1</td>
<td>−5.0000</td>
<td>0.2000</td>
</tr>
<tr>
<td>DMD</td>
<td>2.9500</td>
<td>1</td>
<td>2.9500</td>
<td>0.2400</td>
</tr>
<tr>
<td>Socket, DMD</td>
<td>2.7000</td>
<td>1</td>
<td>2.7000</td>
<td>0.2000</td>
</tr>
<tr>
<td>SUM</td>
<td>+ 0.6500</td>
<td>(1)</td>
<td></td>
<td>0.6400 SUM</td>
</tr>
<tr>
<td></td>
<td></td>
<td></td>
<td></td>
<td>0.3709 RSS</td>
</tr>
<tr>
<td>Max GAP</td>
<td>0.010</td>
<td></td>
<td>1.290</td>
<td></td>
</tr>
<tr>
<td>Min GAP</td>
<td>0.279</td>
<td></td>
<td>1.021</td>
<td></td>
</tr>
</tbody>
</table>

(1) Positive indicates a gap between the interface boss and PCBoard

The height of the boss on the optical chassis was designed to result in a minimum gap of nearly zero for the worst case tolerances. A minimum gap greater than zero is needed to ensure the DMD Datum ‘A’ and its corresponding feature on the optical chassis are in contact. If the PCBoard contacts the boss (on the optical chassis) before the Datum ‘A’ features then focus and focus uniformity could be impacted.
4.3.2 Pin Guide Mounting Concept

The design concept for mounting the Series-450 DMD shown in Figure 21 incorporates specific features on the optical chassis intended to aid DMD alignment during the DMD installation process. While the corresponding features on the optical chassis and those on the DMD (Datum ‘A’, ‘Datum B’, and Datum ‘C’) could be used as the sole alignment aids, these features are small and generally can not be seen during the assembly process. The rough alignment provided by the protruding pin feature is thought to simplify assembly, and reduce the chance for damaging the DMD window due to contact with the Datum ‘B’ and Datum ‘C’ alignment pins (on the optical chassis).

The pin feature shown in Figure 21 is intended to provide an initial rough alignment to a hole in the DMD PCB. The Datum ‘A’, ‘B’, and ‘C’ features (described in earlier sections) are intended to provide the final alignment. The pin on the optical chassis and hole in the PCB together provide a visual alignment aid in addition to being physical aids. Proper placement of the pin is important so as to ensure that any movement of the PCB after the pin and hole are engaged does not result in contact to the DMD window glass from the Datum ‘B’ or ‘C’ pins on the optical chassis.

The detailed tolerance analysis of the critical gaps associated with mounting the heat sink and DMD PCB are the same as for the “Edge Guide” concept described in Section 4.3.1. See Section 4.3.1.1 and Section 4.3.1.2 for more information.

The drawing number for the pin guide mounting concept shown in Figure 21 is 2509479. The 3D-CAD models (in STEP format) and drawings (in pdf format) for each of the parts shown in Figure 21 are available for download, see Section 6.

Figure 21. DMD Mounting Concept – Pin Guide
5 System Sockets

The pins of the Micro-PGA on the Series-450 DMD have the same diameter, length, and spacing as many microprocessor chips. This enables the socket contacts developed for the microprocessor chip sockets to be used for the DMD sockets. The physical contact arrangement used on the DMD is different than those for any microprocessor, therefore the part which holds (and retains) the contacts is unique to the DMD sockets. But the contact design and process for manufacturing the sockets is often the same.

Mechanical outline control envelop specifications are available for both a zero insertion force (ZIF) style and a low insertion force (LIF) style of sockets. The outline specifications define the features which are necessary for the socket to mechanically interface with a Series-450 DMD. Contact design details are unique and proprietary to each socket manufacture, and are therefore omitted from the outline specification. Multiple contact designs from different manufactures will work with the pins on the DMD. Consult the individual socket manufactures for information related to specific socket performance characteristics.

The purpose of the outline envelop specification is to provide a standard set of criteria for development of sockets by different manufacturers. This also enables sockets from different manufacturers to be interchangeable without the need to make any changes to the DMD PCBoard, or the mechanical parts used to mount the DMD. Drawings and 3D-CAD models of the outline envelop specifications are available to DLP® customers and have also been provided to socket manufactures. The 3D-CAD models (in STEP format) and drawings (in pdf format) of both the LIF and ZIF style sockets are available for download, see Section 6.

TI has worked with socket suppliers to develop both ZIF and LIF style sockets. Sockets from these manufactures are available for use.

TI does not perform socket certification or qualification, nor does TI require socket manufactures to conduct any specific suite of tests during development of the sockets. TI encourages socket manufactures to conduct the industry standard tests which exist for these type of sockets (as an indication of the reliability of the socket). The final selection of the socket and manufacture is the choice of each DLP® Customer. It is the responsibility of each customer to verify that a given socket meets the functional and quality requirements for the end-application.

5.1 Socket Design and Selection Considerations

When designing or specifying a socket to be used with the Series 450 DMD the following factors should be considered:

- Style of Socket – both zero insertion force and low insertion force style of sockets have been successfully demonstrated to work with the DMD. Each style has its own characteristics to be considered. See Section 5.3 and Section 5.4 for additional information.

- Installed Height – the Installed Height shown in Figure 22 represents the distance from the top surface of the DMD PCBoard to the "DMD seating plane" on the socket. The "DMD seating plane" is the surface of the socket that will make contact with the DMD and is illustrated in Figure 22. Figure 8 illustrates the "socket seating plane" on the DMD. The socket installed height is one of the key factors when determining the amount the DMD PCBoard could bend if mounted by clamping against bosses on the optical chassis (see the "indirect control" method described in Section 4.2.6.2 and Section 4.3.1.2) .

- Overall X-Y dimensions – The overall socket dimensions influence the foot print area needed on the PCBoard and ultimately the amount of space required for mounting the DMD. ZIF style sockets typically have larger X-Y dimensions. Consequently the DMD mounting hardware must be located further from the DMD than with a LIF style sockets.

- Contact offset – Typically the contacts utilized in a LIF style socket have the center of the (DMD) pin
entry point in-line with the center of the PCBoard solder pad. This is illustrated on the left side of Figure 23 and Figure 24, and Figure 27. Conversely the contacts utilized in a ZIF socket typically have a slight offset between the center of the pin entry point and the center of the solder pad. This is illustrated on the right side of Figure 23 and Figure 24, and Figure 26.

- Single PCBoard Design for both a ZIF and a LIF style socket - A single PCBoard can be designed which accommodates use of both a ZIF and LIF style socket (as shown in Figure 23). When designing a single PCBoard to accommodate both a ZIF and a LIF style socket, it is necessary to accommodate the Contact Offset difference between the ZIF and LIF sockets. This can be accomplished by using slotted mounting holes in the PCBoard. The slots (shown in Figure 23) allow the PCBoard to shift locations such that the optical position of the DMD array can be maintained (utilize the same DMD datums and corresponding features on the optical chassis). Note the shift in the PCBoard edge relative to the fixed screw position (shown in Figure 23) is different between the LIF (dimension Dim-3) and the ZIF (dimension Dim-6) styles. To allow for the PCBoard to move, an open area (available space) is required for the PCBoard assemble to move into. It should be noted that if another electronics assembly is connected to the DMD PCBoard (by a fixed connector) slots would be needed in that.
board as well as an open area to accommodate its shift.

The other option to use both ZIF and LIF style sockets is to design two unique PCBoards (as shown in Figure 24). One for each of the socket styles where the position of the mounting holes and outline is different. Note in Figure 24 the PCBoard edge relative to the fixed screw position (dimension DIM-3) is the same for both the ZIF and LIF styles.

• Electrical Contact Resistance - The electrical contact resistance between the DMD pin and the socket contact should be minimal. A contact resistance of 30 milliohms is a good reasonable objective and should easily be achieved with contact designs available from socket manufactures.

• Contact Insertion Force – the contact insertion force is the force which must be applied to a socket contact by the DMD pin in order to fully seat the pin into a contact (e.g., for the DMD seating plane to make complete contact with the socket seating plane). The contact insertion force multiplied by the total number of pins is the resultant force which must be applied to the DMD in order to fully seat the DMD. The higher the contact insertion force, the more likely that the socket or DMD could become damaged during assembly. As the name implies the ZIF style sockets do not require any insertion force.

• Contact extraction force – the contact extraction force is the force required to overcome the retention forces of the contacts during removal of the DMD from the socket. A socket with a high extraction force is susceptible to damage at the contact solder joint (on the PCBoard pad). A socket with a high extraction force can also contribute to damage of the socket contacts, DMD pins, or DMD ceramic carrier if the DMD is not uniformly removed from the socket (all pins removed at the same time). ZIF style sockets have no extraction force.

When using a non-ZIF style socket, it is recommended that an extraction tool or aid be designed and used to ease removal of the DMD and reduce the possibility of damage to the socket or DMD. Typically an extraction tool will make contact with and press against the PCBoard. Areas on the PCBoard should be defined and identified as "allowable contact areas". These areas should be kept free of both components and surface traces to prevent damage to them from the extraction tool.

• Coefficient of thermal expansion (CTE) – Typically the socket and PCBoard will be made of distinctly different materials and thus different CTE values. DMD sockets are larger than many components which are typically soldered to a PCBoard. Because of the physical size and materials used, a “significant” CTE mismatch is possible between the socket and the PCBoard, which can result in solder joint failures during periods of temperature cycling. What is considered a “significant” CTE mismatch will depend upon the temperature extremes over which your product will operate, but more importantly the solder profile used to attach the socket to the PCBoard. Strict adherence to the solder profiles recommended by the socket manufactures is recommended. Not using the recommended solder profile can result in higher than expected contact insertion and extraction forces.

• Pick and place features - features which facilitate vacuum pickup and placement of the socket onto the PCBoard prior to solder can vary. Some socket suppliers supply a molded cap which is clipped over the contacts; and others manufactures place a piece of kapton tape over the center opening.
5.2 **Socket Nomenclature**

Standard nomenclature has been established to identify DMD Micro-PGA sockets. The nomenclature is derived from the key socket characteristics. Figure 25 illustrates the following nomenclature and associated socket characteristics:

- "Socket 450" - the name which identifies the socket as being designed for use with a Series-450 DMD. Other sockets are available for Series-400 and Series-500 DMDs.
- Installed height – all Socket 450 sockets (both ZIF and LIF style) have an installed height of 2.700 mm. This is important for some methods of mounting the DMD PCBoard to help control loads on the DMD.
- Insertion Force (Style) – indicates whether the socket is a Low or Zero insertion force style socket
- "Opening" - the size of the center opening of the socket. This opening is provided to accommodate access to the DMDs Thermal Interface by a heat sink or other system cooling solution.
- "149" – the number of electrical contacts in the socket. All Series-450 Sockets have 149 contacts. Other Series sockets have different numbers of pins.

**Figure 25. Socket Nomenclature**
5.3 ZIF Style Socket

The TI drawing number for the ZIF style socket envelop design is 2509474.

A 3D-CAD file of the socket nominal geometry (in STEP format) and drawing (in pdf format) are available for download, see Section 6.

The characteristics and considerations of a ZIF style socket are shown in Figure 26 and include:

- larger overall X-Y dimensions (foot print) (compared to a LIF style sockets)
- opening and size corresponding to the location of DMD Thermal Interface area (same for all Series-450 sockets)
- 2.700 installed height (same for all Series-450 sockets)
- 0.3875 mm offset between the center of the DMD pin and PCBoard pad
- no force applied to the DMD when inserting or removing the DMD
- extraction tool for the DMD is not needed
- insertion tool for DMD is most likely not needed

Figure 26. Zero Insertion Force Style Series-450 Socket
5.4 **LIF Style Socket**

The TI drawing number for the LIF style socket envelop design is 2509473.

A 3D-CAD file of the socket nominal geometry (in STEP format) and drawing (in pdf format) are available for download, see Section 6.

The characteristics and considerations of a LIF style socket are shown in Figure 27 and include:

- smallest overall X-Y dimensions (footprint)
- opening and size corresponding to the location of DMD Thermal Interface area (same for all Series-450 sockets)
- 2.700 installed height (same for all Series-450 sockets)
- no offset between the center of the DMD pin and PCBoard pad
- force must be applied to the DMD when installing the DMD into the socket
- extraction tool for the DMD may be desired when removing the DMD from the socket to assure uniform removal (and avoid possible damage to the socket or DMD)
- contains features to allow for use of an extraction tool
- insertion tool for DMD may be desired
- Note that use of an extraction tool would most likely increase the effective board footprint size (to allow for tool contact areas on the PCBoard). The increase would most likely be in the 22.30 dimension direction only (adjacent to the edge recesses). Any tool contact areas on the PCBoard must be component and trace keep out areas to avoid damage to traces or vias.

![Figure 27. Low Insertion Force Style s450 Socket](image-url)
Drawings (in PDF format) and 3D-CAD models (in STEP format) of many of the parts discussed in this application note are available to facilitate study when designing an end-application. Table 6 summarizes the literature numbers for the drawings and 3D-CAD models that are available for download.

Table 6. Reference Drawings and 3D-CAD Models

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<td>DLPC049</td>
<td>3D-CAD and drawing files of Edge Guide mounting concept (2509652)</td>
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<td>DLPC052</td>
<td>3D-CAD and drawing files of LIF style socket (2509473)</td>
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